

The influence of an abrupt convergence on heat transfer in circular ducts: a theoretical assessment

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This is a theoretical assessment of the postulation made by Barozzi *et al*, of separation in laminar flow in a circular duct caused by an abrupt entry. Using sets of data supplied by Barozzi, predictions have been made using a program developed by Collins. The effects of viscosity variation with temperature, and of natural convection, are quantified for the flow of water in a vertical tube. The results are consistent with the discussion of Barozzi *et al*, and tend to confirm the presence of a definite abrupt entry effect

Key words: laminar flow, entry effects, separation, heat transfer

In an earlier study by the author¹, it was suggested that the presence of an abrupt, as opposed to a smoothed, entry to a circular duct, could affect a laminar flow-field. The recent experiments of Barozzi *et al*² indicate that heat transfer could also be affected over some distance from entry, and they postulate that separation has probably occurred. In reaching this conclusion, however, they compare their heat transfer values with analytical predictions of Churchill and Ozoe³ which assume constant properties. For vertical flow of water, both viscosity variation with temperature and natural convection may be significant, and Barozzi *et al* make informed assessments of these effects on heat transfer. Both the effects, however, are accommodated within the laminar flow treatment of Collins⁴. This permits the separation hypothesis to be assessed with more confidence by accurately quantifying these effects. To this end typical experimental data were supplied by Barozzi⁵, so that corresponding sets of theoretical predictions could be made.

Theoretical assessment

Seven sets of data (runs) were supplied by Barozzi⁵ and defined† as in Table 1 in order of increasing Re .

Using the program developed by Collins^{1,4}, predictive runs were made for each experimental run, assuming a uniform inlet velocity profile. The following schemes were used:

(A) Constant properties, with Re and Pr set at their inlet values,

(B) As (A), but allowing μ to vary with temperature,

(C) As (B), but allowing ρ to vary in addition, ie a combined convection case.

Results for selected runs 1, 3, 4 and 7, and all schemes, are plotted in Figs 1(a)–(d) respectively, in the same manner as Barozzi *et al*'s² Figs 3–6. The experimental data are also shown. The comparisons are increasingly consistent at downstream distances and serve to confirm and clarify several points discussed in the paper.

As shown in Table 2, the predicted effect of viscosity variation is to increase Nu at the end of the

Table 1 Definition of sets of experimental data

Run	Mean Re	Mean Pr	Inlet temp, °C	Heat flux, W/m^2
1	768	4.236	37.31	5.578×10^3
2	845	5.848	25.07	3.604×10^3
3	1234	6.216	20.7	1.081×10^4
4	1308	4.606	35.18	5.538×10^3
5	1601	6.447	20.23	1.081×10^4
6	1705	6.471	20.28	1.081×10^4
7	1796	4.258	37.73	1.074×10^4

Table 2 Predicted effect of viscosity and density on heat transfer at 80 diameters

Run	Predicted increase in Nu %	
	Due to μ	Due to ρ
1	3.93	39.48
2	2.78	7.35
3	6.76	10.70
4	3.29	11.77
5	6.38	6.35
6	6.28	5.64
7	5.20	18.55

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† The notation is consistent with that of Barozzi *et al* on page 46 of this issue

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heated length by between about 3 and 7%. Again as shown in Table 2, the predicted effect of free convection is similarly to increase Nu by between about 6 and 18%. Run 1, with an increase of about 40% is an exception with a strong buoyancy effect, and Fig

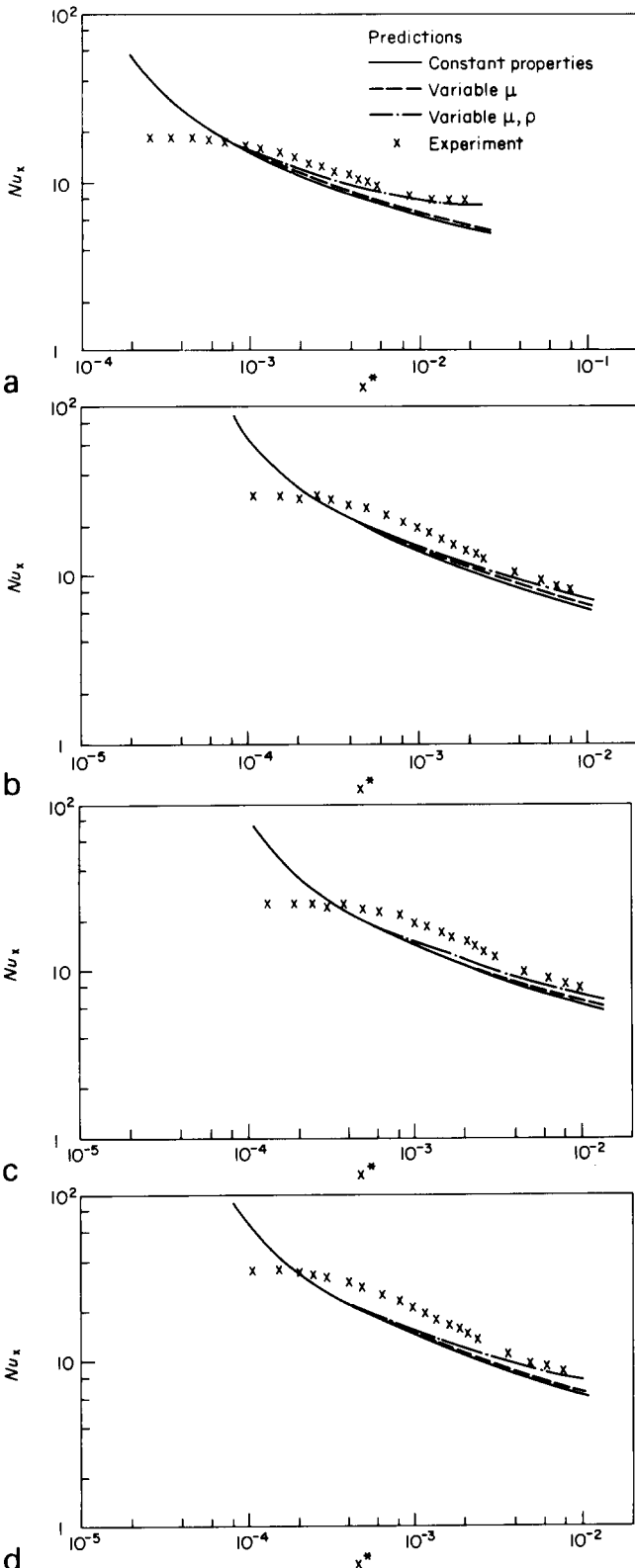


Fig 1 Heat transfer comparisons (a) Run 1—Re: 768, Pr: 4.2355; (b) Run 3—Re: 1234, Pr: 6.2158; (c) Run 4—Re: 1308, Pr: 4.6061; and (d) Run 7—Re: 1796, Pr: 4.2583

1(a) shows that both experiment and prediction 'level off' in a very similar manner.

The analysis allows for axial conduction, which under these conditions is negligible (see Collins⁶). Finally, and perhaps to be expected, the above effects are indeed negligible near the inlet.

Now for a constant wall heat flux, the only variable determining Nu is the wall temperature, and Table 3 gives a comparison of these, three-quarters of the distance along the duct. The predictions overestimate the temperatures by only about 2.5°C maximum and this (particularly Run 1) is completely comparable to results in Collins *et al*⁴ for upward flow of oil with a sharp-edged entry. These relate to experimental data of Kemeny and Somers⁷, and are given in Table 4. Initially, there is an underprediction (Nu higher), followed by a consistent, but small, overprediction of temperature.

Conclusions

The results presented here confirm the assessment of the effects of viscosity variation and natural convection made by Barozzi *et al*. They thus confirm the basic premise of a definite sharp-edged entry effect on heat transfer under the experimental conditions. Also, such an effect may be quantified more accurately by comparing experimental data with scheme (C) type predictions.

Acknowledgement

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Table 3 Comparison of wall temperature and Nu

Run	Experiment‡		Predictions‡	
	Wall temp, °C	Nu	Wall temp, °C	Nu
1	55.3	7.52	56.05	7.206
2	36.3	6.96	37.15	6.384
3	47.3	8.06	49.51	7.638
4	49.9	7.73	51.30	7.024
5	44.0	8.73	46.38	8.099
6	43.2	8.99	45.87	8.227
7	62.2	8.59	64.19	8.042

‡ Experiment, 59.56 dia; prediction, 60 dia

Table 4 Wall temperature comparison

Distance from entry, dia.	Wall temperature, °C	
	Experiment ⁷	Prediction ⁴
2.67	52.61	51.67
10.00	62.06	62.36
17.35	66.39	67.22
24.67	69.28	70.50
32.00	71.81	73.22

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